

**SUBSTITUTE SPECIFICATION****METHOD AND FEED DEVICE FOR EFFECTING THE ADVANCE MOVEMENT OF  
AT LEAST ONE TOOL SUPPORT THAT ROTATES AROUND A ROTATIONALLY  
SYMMETRICAL PART**

[0001] This application is the national phase under 35 U.S.C. § 371 of PCT International Application No. PCT/DE00/01980 which has an International filing date of June 20, 2000, which designated the United States of America, the entire contents of which are hereby incorporated by reference.

**Field of the Invention**

[0002] The present invention generally relates to working machines, and more particular, feed devise that act on components.

**Background of the Invention**

[0003] The machining of large machine shafts, for example turbine and generator shafts, requires special lathes for which large machine sheds are necessary. For working on site, for example for machining the bearing seats during repair by lathe-turning, grinding or polishing, it would be desirable to carry out machining by way of nonstationary machines which can be placed onto the stationary shaft, so that extensive and costly demounting and transport work could be avoided. Even possible damage to the shaft as a result of transport to the place of use itself and also during subsequent installation work often make additional work on the shaft desirable.

[0004] For uses on smaller shafts, lathes are already known which are placed onto a shaft and rotate around the shaft. The problem of such machining appliances is the generation of advance movements for the machining tools in a longitudinal direction and transversely to the shaft. Separate drives which corotate with the lathe have to be used for generating the advancing movements. Apart from the large mass which has to be moved and supported in this way, thus causing vibrations which lead to machining inaccuracies, such an arrangement has further disadvantages. The energy for these drives has to be transmitted to the rotating drives by way of slip rings. The working machine must therefore have an unsplit design, since the slip rings would otherwise also have to be split, but power transmission via split slip rings would present considerable technical difficulties.

[0005] It is desirable, by contrast, for a working machine to be designed to be splittable in order to be placed onto a correspondingly large shaft and for the working machine to be assembled for a machining operation on the shaft, though without requiring power

transmission via split slip rings. Moreover, the rotating mass is to be kept as small as possible.

[0006] A feed device is disclosed in JP-A 62287907, in which the drives for the advancing movements are likewise arranged at a fixed location. The advancing movements take place via the relative movement of a further transmission mechanism cooperating with the respective leadscrew, in addition to the main transmission mechanism. These transmission mechanisms are connected to the main transmission mechanism in each case via a planetary gear, the planet wheels of which can be additionally driven or braked by rest motors and consequently bring about the relative movement. The solution has the disadvantage that planetary gears of this type are highly cost-intensive, and that, particularly in the case of run-on and run-off ramps, internal forces occur which lead to unintended relative movements of the main drive and advancing drive and consequently to unintended adjustments of the machining tools.

#### SUMMARY OF THE INVENTION

[0007] An object on which the present invention is based is to specify a method for carrying out the advancing movement and a feed device for a working machine with a rotating tool support the method and the feed device allowing machining (lathe-turning, milling, orbital grinding) in NC quality.

[0008] The object is achieved, according to the present invention, by way of the features in at least claims 1 and 3. Expedient refinements of the present invention are contained in the other claims.

[0009] The advancing movement of the tools is generated in that transmission mechanisms are provided, which drive the leadscrews of the tool supports and are themselves driven at a different speed from the working machine which rotates as a whole; the relative speed between the two then takes effect.

[0010] If the relative speed is zero, no advancing movement takes place.

[0011] Preferably, a plurality of large gear rings are provided as transmission mechanisms between the drive motors and the working machine. One of these gear rings is provided for rotating the entire working machine and consequently also determining the cutting speed of a tool with respect to the shaft, while the other gear ring, or gear rings, serves for the advancing movements of the tools.

[0012] When all the gear rings rotate at the same speed, no movement takes place on the cross slide and longitudinal slide supports. Only when the gear rings for the supports for longitudinal and cross movement run more quickly or more slowly than the gear ring driving the working machine, is there a rotation of the leadscrews and therefore an advance of the

supports in relation to the working machine on account of the relation movement between the gear rings. It is therefore necessary to drive the rest motors, which act on the individual gear rings, more quickly or more slowly than the gear ring for the working machine when an advancing movement is to be brought about.

[0013] The gear rings are preferably provided with the same number of external teeth. The pinions driving the gear rings must then have identical diameters. The gear rings for driving the leadscrews are toothed internally and drive the leadscrews of the rests via pinions which are mounted in a rotationally movable manner in the rotating working machine.

[0014] There is a drive motor in each case for driving the gear ring of the rotating working machine and that of the leadscrews. In order to drive all the gear rings synchronously with the main motor in an operating phase in which no advancing movement is to take place, according to the present invention, a mechanical coupling of the main motor to the support motors is provided; for example, via toothed belts, via which the housings of the rest motors are taken up by the main motor. Specifically, in the case described here, at the same rotational speed, the shafts of the rest motors, which are not themselves driven in this operating phase, also being taken up at the same rotational speed and driving the gear rings for the leadscrew movement, so that there is no movement of these gear rings in relation to the gear ring which brings about the rotational movement of the entire working machine. In order at the same time to rule out the effect of internal forces in the working machine, each support motor is expediently braked in this operating phase.

[0015] All the drive motors are arranged at a fixed location, for example directly on the stationary shaft or on a block standing next to the shaft. In this case, the tools controlled by the leadscrews rotate together with the working machine, without executing a relative movement perpendicularly to or along the shaft, as long as the rotational speed of the motor shaft of one or both support motors is not changed in relation to the rotational speed of the driving main motor. Only when, as a result of the switched-on specific drive movement of one of the rest motors, one leadscrew or another rotates more quickly or more slowly than the working machine, about the shaft to be machined, is there a movement of the tool or tools in relation to the shaft. The supply of power to the support motors, the housings of which are fixed in place, but rotate at the same rotational speed as the main motor, takes place via slip rings. The voltage supplied via the slip rings determines the rotational speed of the respective support motor and consequently the advancing speed.

[0016] Instead of lathe tools, other tools may also be used for machining the shaft surface, such as, for example, grinding wheels, milling cutters or polishing devices.

[0017] It is assumed above that all the gear rings and the pinions driving the gear rings have the same diameters and numbers of teeth, and the main motor and the housings of the support

motors have the same rotational speed. This is certainly the most practical solution. It is also possible, however, to use gear rings with different diameters, when the pinions likewise have different diameters and/or the rotational speeds of the motors are not identical. It is important merely that, in the operating state in which no advancing movement is to take place, all the gear rings are driven at the same rotational speed.

[0018] The solution has the advantage that the working machine may have a split design. All the drive motors are to be arranged at a fixed location, and the support motors do not corotate with the entire machine, but only independently. The rotating mass is therefore also kept small. The power transmission to the rest motors may be carried out via unsplit slip rings. A simple and accurate control of the tool supports becomes possible, even in the case of run-on and run-off ramps, and during curve machining of a component to be machined. That is to say surfaces, diameters and curves may be machined, programmed, by way of the NC technique, as in conventional machine tools.

#### BRIEF DESCRIPTION OF THE DRAWINGS

[0019] The invention will be explained in more detail below with reference to an exemplary embodiment. In the accompanying drawings:

Fig. 1 shows a diagrammatically illustrated side view of a working machine according to the present invention, and

Fig. 2 shows a top view of the driving side of the working machine according to fig. 1.

#### DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENTS

[0020] For the sake of clarity, the exemplary embodiment is preferably an arrangement with two gear rings. That is to say, one gear ring for driving the working machine, and one gear ring for driving a leadscrew for the longitudinal advancing movement of a support. In a practical embodiment of the present invention, at least one third gear ring will be provided for carrying out a second advancing movement of the tool. The embodiments described herein are hereby illustrated in the present invention.

[0021] A working machine is placed onto a shaft 1 to be machined and is supported in a rotationally movable manner on the latter by way of bearings 2. The working machine includes a frame support 3, in which is mounted a tool support 4 that may execute an advancing movement in the longitudinal direction of the shaft 1 via a leadscrew 5. The frame support 3 is designed on the driving side as an externally toothed gear ring 6. Via a pinion 7 on the motor shaft of a main motor 8, the gear ring 6, and therefore also the tool support 4, is driven, so that the latter rotates together with its tool 9, for example a lathe tool, about the

shaft 1. The main motor 8 is in this case fastened on the shaft 1 via a support 10.

[0022] The leadscrew 5 is rotatable in the frame support 3 via a pinion 11 and moves the tool rest 4 back and forth via a worm drive. The pinion 11 is itself driven by an internally an externally toothed gear ring 12 which is mounted rotatably on the frame support 3. As long as the gear rings 6 and 12 do not execute any movement in relation to one another, the tool rest 4 remains in its position, that is to say no advance is brought about. For this purpose, assuming the same number of teeth of a pinion 17 and pinion 7 and of the gear rings 6 and 12, the pinion 17 must be driven at the same rotational speed as the pinion 7. This is carried out by the housing of a support motor 13, on the motor shaft of which the pinion 17 is arranged, being mounted rotatably and rotating with the same rotational speed as the main motor 8 or the pinion 7, the housing taking up the pinion 11 at this rotational speed. In order to bring about rotation and at the same time ensure full synchronism of the two drive movements, the main motor 8 is coupled mechanically to the housing of the support motor 13 via a toothed belt 14.

[0023] In order, in an operating phase without an advancing movement, not to allow any rotational speed of the pinion 17 which deviates from the rotational speed of the housing of the rest motor 13, the rest motor 13 is expediently braked, so that the housing and the motor shaft of the support motor 13 are coupled. In contrast to this, for an advancing movement of the tool rest 4, the brake is released and the rest motor 13 is additionally driven itself. This takes place via the supply of power to slip rings 15 on the rest motor 13. When the rest motor 13 is put into operation, the pinion 17, and consequently the gear ring 12, is additionally driven in one direction or braked in the other direction beyond the rotation that is imparted by the housing of the rest motor 13. A movement of the gear rings 6 and 12 in relation to one another thus takes place, these gear rings bringing about a rotation of the leadscrew 5 and consequently an advance of the tool rest 4.

[0024] Since the rest motor 13, together with its slip ring set, is arranged at a fixed location, the frame support 3 can have a split design, so that it can be placed onto the shaft 1 anywhere on the latter.

[0025] As can easily be seen, a further advancing movement radially relative to the shaft 1 can be achieved by way of a second support motor and a third gear ring and also a conventional deflection mechanism on the tool rest. If further rest drives are also necessary, these can be implemented in the same way.

[0026] The invention being thus described, it will be obvious that the same may be varied in many ways. Such variations are not to be regarded as a departure from the spirit and scope of the invention, and all such modifications as would be obvious to one skilled in the art are intended to be included within the scope of the following claims.

Description

Method and feed device for <sup>effecting</sup> ~~carrying out~~ the <sup>advance</sup> ~~advancing~~ movement of at least one tool ~~rest~~ <sup>rotating</sup> about a  
 5 rotationally ~~symmetric component~~ <sup>support that rotates around</sup> ~~symmetrical part~~

Field of the invention

Background of  
the invention

10 The machining of large machine shafts, for example turbine and generator shafts, requires special lathes for which large machine sheds are necessary. For working on site, for example for machining the bearing seats during repair by lathe-turning, grinding or polishing, it would be desirable to carry out machining by <sup>way</sup> ~~means~~ of nonstationary machines which can be placed onto the stationary shaft, so that extensive and costly  
 15 demounting and transport work could be avoided. Even possible damage to the shaft as a result of transport to the place of use itself and also during subsequent installation work often make additional work on the shaft ~~seem~~ desirable.

20

For uses on smaller shafts, lathes are already known which are placed onto a shaft and rotate <sup>around</sup> ~~about~~ the shaft. The problem of such machining appliances is the generation of <sup>advance</sup> ~~advancing~~ movements for the machining  
 25 tools in a longitudinal direction and transversely to the shaft. Separate drives which corotate with the lathe have to be used for generating the advancing movements. Apart from the large mass which has to be moved and supported in this way, thus causing  
 30 vibrations which lead to machining inaccuracies, such an arrangement has further disadvantages. The energy for these drives has to be transmitted to the rotating drives by <sup>way</sup> ~~means~~ of slip rings. The working machine must therefore have an unsplit design, since the slip rings  
 35 would otherwise also have to be split,

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but power transmission via split slip rings would present considerable technical difficulties.

It is desirable, by contrast, for a working machine to be designed to be splittable in order to be placed onto a correspondingly large shaft and for the working machine to be assembled for a machining operation on the shaft, though without requiring power transmission via split slip rings. Moreover, the rotating mass is to be kept as small as possible.

*is disclosed in*  
 UP-A 62287907 ~~has already disclosed~~ *↑* a feed device in which the drives for the advancing movements are likewise arranged at a fixed location. The advancing movements take place via the relative movement of a further transmission mechanism cooperating with the respective leadscrew, in addition to the main transmission mechanism. These transmission mechanisms are connected to the main transmission mechanism in each case via a planetary gear, the planet wheels of which can be additionally driven or braked by rest motors and consequently bring about the relative movement. The solution has the disadvantage that planetary gears of this type are highly cost-intensive, and that, particularly in the case of run-on and run-off ramps, internal forces occur which lead to unintended relative movements of the main drive and advancing drive and consequently to unintended adjustments of the machining tools.

*Summary of the Invention*  
 The ~~an~~ object on which the *Present* invention is based is to specify a method for carrying out the advancing movement and a feed device for a working machine with a rotating tool *support* ~~rest~~, *the* said method and ~~said~~ *the* feed device allowing machining (lathe-turning, milling, orbital grinding) in NC quality.

AMENDED SHEET

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The object is achieved, according to the <sup>present</sup> invention, by <sup>at least</sup> means of the features in the characterizing part of claims 1 and 3 ~~in conjunction with the features in the preamble~~. Expedient refinements of the invention are  
5 contained in the ~~subclaims~~ <sup>other claims</sup> <sup>present</sup>.

The advancing movement of the tools is generated in that transmission mechanisms are provided, which drive the leadscrews of the tool <sup>supports</sup> ~~rests~~ and are themselves  
10 driven at a different speed from the working machine which rotates as a whole; the relative speed between the two then takes effect.



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If the relative speed is zero, no advancing movement takes place.

Preferably, a plurality of large gear rings are provided  
 5 as transmission mechanisms between the drive motors and  
 the working machine. <sup>Y</sup> One of these gear rings ~~being~~ <sup>is</sup>  
 provided for rotating the entire working machine and  
 consequently also determining the cutting speed of a  
 tool with respect to the shaft <sup>Y</sup> while <sup>A</sup> the other gear  
 10 ring, or gear rings, serves ~~or serve~~ for the advancing  
 movements of the tools.

When all the gear rings rotate at the same speed, no  
 movement takes place on the cross slide and  
 15 longitudinal slide ~~rests~~ <sup>supports</sup>. Only when the gear rings for  
 the ~~rests~~ <sup>supports</sup> for longitudinal and cross movement run more  
 quickly or more slowly than the gear ring driving the  
 working machine, <sup>is</sup> there a rotation of the leadscrews  
 and therefore an advance of the ~~rests~~ <sup>supports</sup> in relation to  
 20 the working machine on account of the relation movement  
 between the gear rings. It is therefore necessary to  
 drive the rest motors, which act on the individual gear  
 rings, more quickly or more slowly than the gear ring  
 for the working machine when an advancing movement is  
 25 to be brought about.

The gear rings are preferably provided with the same  
 number of external teeth. The pinions driving the gear  
 rings must then have identical diameters. The gear  
 30 rings for driving the leadscrews are toothed internally  
 and drive the leadscrews of the rests via pinions which  
 are mounted in a rotationally movable manner in the  
 rotating working machine.

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There is a drive motor in each case for driving the gear ring of the rotating working machine and that of the leadscrews. In order to drive all the gear rings synchronously with the main motor in an operating phase

5 in which no advancing movement is to take place, according to the <sup>present</sup> invention, a mechanical coupling of the main motor to the ~~rest~~ <sup>support</sup> motors is provided, for example, via toothed belts, via which the housings of the rest motors are taken up by the main motor. Specifically, in

10 the case described here, at the same rotational speed, the shafts of the rest motors, which are not themselves driven in this operating phase, also being taken up at the same rotational speed and driving the gear rings for the leadscrew movement, so that there is no

15 movement of these gear rings in relation to the gear ring which brings about the rotational movement of the entire working machine. In order at the same time to rule out the effect of internal forces in the working machine, each ~~rest~~ <sup>support</sup> motor is expediently braked in this

20 operating phase.

All the drive motors are arranged at a fixed location, for example directly on the stationary shaft or on a block standing next to the shaft. In this case, the

25 tools controlled by the leadscrews rotate together with the working machine, without executing a relative movement perpendicularly to or along the shaft, as long as the rotational speed of the motor shaft of one or both ~~rest~~ <sup>support</sup> motors is not changed in relation to the

30 rotational speed of the driving main motor. Only when, as a result of the switched-on specific drive movement of one of the rest motors, one leadscrew or another rotates more quickly or more slowly than the working machine, about the shaft to be machined, is there a

35 movement of the tool or tools in relation to the shaft. The supply of power to the ~~rest~~ <sup>support</sup> motors, the housings of which are fixed in place, but rotate at the same rotational speed as the main motor, takes place via slip rings.

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<sup>determines</sup>  
 The voltage supplied via the slip rings ~~determining~~ the rotational speed of the respective ~~rest~~ motor and consequently the advancing speed. <sup>support</sup>

- 5 Instead of lathe tools, other tools may also be used for machining the shaft surface, such as, for example, grinding wheels, milling cutters or polishing devices.

- 10 It <sup>is</sup> ~~was~~ assumed above that all the gear rings and the pinions driving <sup>the gear rings</sup> ~~these~~ have the same diameters and numbers of teeth, and the main motor and the housings of the <sup>support</sup> ~~rest~~ motors have the same rotational speed. This is certainly the most practical solution. It is also possible, however, to use gear rings with different  
 15 diameters, when the pinions likewise have different diameters and/or the rotational speeds of the motors are not identical. It is important merely that, in the operating state in which no advancing movement is to take place, all the gear rings are driven at the same  
 20 rotational speed.

- The solution has the advantage that the working machine <sup>may</sup> ~~can~~ have a split design. All the drive motors are to be arranged at a fixed location, and the <sup>support</sup> ~~rest~~ motors do  
 25 not corotate with the entire machine, but only independently. The rotating mass is therefore also kept small. The power transmission to the rest motors <sup>may</sup> ~~can~~ be carried out via unsplit slip rings. A simple and accurate control of the tool <sup>supports</sup> ~~rests~~ becomes possible,  
 30 even in the case of run-on and run-off ramps, and during curve machining of <sup>a</sup> ~~the~~ component to be machined <sup>y</sup> ~~x~~ that is to say surfaces, diameters and curves <sup>may</sup> ~~can~~ be machined, programmed, by <sup>way</sup> ~~means~~ of the NC technique, as in conventional machine tools.

# BRIEF DESCRIPTION OF THE <sup>DRAWINGS</sup> - 6 -

The invention will be explained in more detail below with reference to an exemplary embodiment. In the accompanying drawings:

- fig. 1 shows a diagrammatically illustrated side view  
 5 of a working machine according to the present invention, and  
 fig. 2 shows a top view of the driving side of the working machine according to fig. 1.

## DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENTS

- 10 For the sake of clarity, the exemplary embodiment is ~~restricted to~~ <sup>pertaining to</sup> an arrangement with two gear rings <sup>gear ring</sup> that is to say, one <sup>gear ring</sup> for driving the working machine, ~~as such~~ and one <sup>gear ring</sup> for driving a leadscrew for the longitudinal advancing movement of a ~~rest~~ <sup>support</sup>. In ~~the~~ <sup>a</sup> practical ~~version~~ <sup>embodiment</sup> of ~~the present~~ <sup>invention</sup>  
 15 at least one third gear ring will be provided for carrying out a second advancing movement of the tool. ~~The~~ <sup>The</sup> ~~embodiments~~ <sup>embodiment</sup> described herein are merely illustrated of the present invention  
 20 ~~The~~ <sup>A</sup> working machine is placed onto a shaft 1 to be machined and is supported in a rotationally movable manner on the latter by ~~means~~ <sup>way</sup> of bearings 2. ~~It~~ <sup>The working machine</sup> ~~includes~~ <sup>includes</sup> ~~of~~ <sup>of</sup> a frame support 3, in which is mounted a tool ~~rest~~ <sup>support</sup> 4 which can execute an advancing movement in the longitudinal direction of the shaft 1 via a leadscrew 5. The frame support 3 is designed on the  
 25 driving side as an externally toothed gear ring 6. Via a pinion 7 on the motor shaft of a main motor 8, the gear ring 6, and therefore also the ~~tool rest~~ <sup>support</sup> 4, is driven, so that the latter rotates together with its tool 9, for example a lathe tool, about the shaft 1.  
 30 The main motor 8 is in this case fastened on the shaft 1 via a support 10.

The leadscrew 5 is rotatable in the frame support 3 via a pinion 11 and moves the tool rest 4 back and forth  
 35 via a worm drive. The pinion 11 is itself

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driven by an internally an externally toothed gear ring 12 which is mounted rotatably on the frame support 3. As long as the gear rings 6 and 12 do not execute any movement in relation to one another, the tool rest 4 remains in its position, that is to say no advance is brought about. For this purpose, assuming the same number of teeth of ~~the~~<sup>a</sup> pinion 17 and pinion 7 and of the gear rings 6 and 12, the pinion 17 must be driven at the same rotational speed as the pinion 7. This is carried out by the housing of a ~~rest~~<sup>support</sup> motor 13, on the motor shaft of which the pinion 17 is arranged, being mounted rotatably and rotating with the same rotational speed as the main motor 8 or the pinion 7, ~~said~~<sup>the</sup> housing taking up the pinion 11 at this rotational speed. In order to bring about rotation and at the same time ensure full synchronism of the two drive movements, the main motor 8 is coupled mechanically to the housing of the ~~rest~~<sup>support</sup> motor 13 via a toothed belt 14.

20 In order, in an operating phase without an advancing movement, not to allow any rotational speed of ~~its~~ <sup>the</sup> pinion 17 which deviates from the rotational speed of the housing of the rest motor 13, the rest motor 13 is expediently braked, so that the housing and the motor shaft of the ~~rest~~<sup>support</sup> motor 13 are coupled. In contrast to this, for an advancing movement of the tool rest 4, the brake is released and the rest motor 13 is additionally driven itself. This takes place via the supply of power to slip rings 15 on the rest motor 13. When the rest motor 13 is put into operation, the pinion 17, and consequently the gear ring 12, is additionally driven in one direction or braked in the other direction beyond the rotation ~~which~~<sup>that</sup> is imparted by the housing of the rest motor 13. A movement of the gear rings 6 and 12 in relation to one another thus takes place, these gear rings bringing about a rotation of the leadscrew 5 and consequently an advance of the tool rest 4.

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Since the rest motor 13, together with its slip ring set, is arranged at a fixed location, the frame support 3 can have a split design, so that it can be placed onto the shaft 1 anywhere on the latter.

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As can easily be seen, a further advancing movement radially relative to the shaft 1 can be achieved by <sup>way</sup>~~means~~ of a second rest motor and a third gear ring and also a conventional deflection mechanism on the tool rest. If further <sup>super</sup>~~rest~~ drives are also necessary, these can be implemented in the same way.

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Variations PP